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(54) **Whistle**

Pfeife

Sifflet

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EP 0 930 605 B1

Description

[0001] This invention relates to a whistle and is more particularly concerned with a dog whistle.

[0002] Dog whistles are very well known. Nevertheless, it is believed that all operating parameters have not been optimised in a single design and that there still tends to be a number of problems associated with such whistles that, as yet, have not been overcome.

[0003] Dog whistles having a single sound chamber and having a good volume of sound to be produced at a high frequency effective for dogs tend to be larger than need necessarily be the case. Additionally, such whistles need to be able to be blown quite gently for close distance work with the dog but also should be capable of being blown very hard for longer distance work. Furthermore, during the act of blowing, the sound frequency should not fluctuate unduly or otherwise this may lead to confusion on the part of the dog regarding the particular operating command.

[0004] Patent publication GB 468,821 discloses an example of such a dog whistle, whilst patent publication DE 258,968 discloses a more general type of whistle.

[0005] A well known problem with this type of whistle is in achieving a compromise in the design, in order to successfully achieve both the aforementioned aims of enabling the whistle to be blown gently and effectively for close distance work without the whistle becoming very quiet when over-blown and thus losing effectiveness at long distances. Where such a whistle is designed to be very effective when blown hard it may simply fail to respond to gentle blowing and thus may be excellent for distance work but practically useless for close work.

[0006] It is an object of the present invention to at least alleviate one of the aforementioned problems associated with whistles and/or to provide a whistle which is improved in at least some respect.

[0007] Canadian patent specification No.CA-A-2166048 shows a whistle-like instrument having a type of mouthpiece used on a recorder and an elongate body having a bore and a movable baffle within said bore like the whistle of GB 468,821 mentioned before a movable baffle within. In one position of the baffle significant changes in the pitch of the wailing sound can be made by varying the pressure of air blown into the instrument to replicate the call of a common loon. In response of different pressure of air blown through the instrument a jump in frequency from 800 hertz to 1000 is achieved which is well within a 400 Hz bandwidth. In addition it should be noted that a mouthpiece as used on a recorder (like eg. is disclosed by patent publication GB 1,479,286) is known to have very specific constructional dimensions and relationships.

[0008] According to the present invention there is provided a whistle as set forth in claim 1.

[0009] Preferably, the sound chamber is shaped and dimensioned to produce sound over a narrow frequency

bandwidth lying in the range of about 400 to about 800 Hertz and preferably the bandwidth is about 400 Hertz \pm 20 Hertz.

[0010] The sound chamber is generally cylindrical in shape and its diameter is within about 10 percent of the axial length of the chamber.

[0011] The overall length of the whistle is preferably about 5.3975 centimetres \pm 0.5 centimetre. The overall width or diameter of the whistle is preferably about 1.473 centimetres \pm .25 centimetres.

[0012] The whistle of the present invention is provided, in the air passage, with means (usually in the form of a ramp) for directing the air flow blown into the mouthpiece towards the sound producing means which is in the form of an air splitter, said air flow directing means having an operative surface which is inclined at an angle in the range of about 3° 30' to about 5° and may optimally be 4° 30' \pm 10 minutes. This angle is much reduced compared with known whistles of the kind referred to and generally should enable compression of the air blown into the mouthpiece to be reduced as it impinges on the air splitter. This is important as it guarantees the highest possible air pressure on the air splitter. Additionally, a more even distribution of air may be provided with the added advantage of preventing some frequency fluctuation.

[0013] The air splitter is formed by an edge of a surface arranged at an angle which lies within a range of about 45° to about 55° with respect to a transverse plane perpendicular to the axis of the sound chamber. This angle may optimally be 55°, and provides a marked improvement in performance over known whistles, effectively reducing any back pressure in the sound chamber which can create stalling and silencing of the whistle under pressure.

[0014] Where air flow directing means is provided as aforesaid, the operative surface of such means may be provided at the end thereof adjacent the sound chamber with a face which is inclined to the axis of the air passage at an angle 45° \pm 5°. Correctly selecting this angle may create a more constant frequency throughout the same range of the whistle.

[0015] The length of said face may be .127 centimetres \pm .05 centimetres. Any length longer than this would cause stalling of the whistle through the build-up of back pressure while any shorter length would allow the frequency to vary as the blowing pressure is varied. The overall length of the operative surface of the air directing means and this face may be of the order of 1.98 centimetres.

[0016] Many advantageous features of the whistle in accordance with the invention will be evident from the following description and drawings.

[0017] An embodiment of a dog whistle in accordance with the present invention will now be described, by way of example only, with reference to the accompanying drawings, in which:

FIGURE 1 shows a side view of the whistle which is formed in two parts;

FIGURE 2 shows a top view of the whistle shown in FIGURE 1;

FIGURE 3 shows an end view of the whistle looking in the direction of arrow "A" in FIGURE 1;

FIGURE 4 is a side view of the lower part of the whistle showing it in an inverted position;

FIGURE 5 is an underneath plan view corresponding to FIGURE 4;

FIGURE 6 is a longitudinal section through the lower part of the whistle;

FIGURE 7 shows an end view looking in the direction of arrow "B" in FIGURE 5;

FIGURE 8 is a cross section taken on line VIII-VIII of FIGURE 5;

FIGURE 9 shows an enlarged detail of the part of the whistle sound chamber defined by the lower part;

FIGURE 10 shows a detail of a lip provided around the part of the sound chamber shown in FIGURE 9;

FIGURE 11 shows in detail one of five locating pegs provided on the lower part of the whistle;

FIGURE 12 shows a side view of the upper part of the whistle;

FIGURE 13 is an underneath plan view of the upper part of the whistle shown in FIGURE 12;

FIGURE 14 is a longitudinal section through the upper part of the whistle in an inverted position;

FIGURE 15 is a cross section taken on the line X'-X' of FIGURE 13;

FIGURE 16 is a cross-section taken on the line Y-Y of FIGURE 13;

FIGURE 17 shows an enlarged detail of the part of the sound chamber of the whistle which is formed in the upper part of the whistle;

FIGURE 18 shows a detail of a groove provided around the part of the sound chamber shown in FIGURE 17; and

FIGURE 19 shows in detail one of five holes provided

in the upper part of the whistle.

[0018] The drawings show a very compact dog whistle 1 comprising a body formed in two parts, referred to as an upper part 2 (see particularly FIGURES 12 to 16) and a lower part 3 (see particularly FIGURES 4 to 8). Thus a notional split line X-X is shown in FIGURE 1 separating the upper part 2 from the lower part 3. The upper part 2 has five locating pegs p (see FIGURES 4 to 8 and FIGURE 11) engageable during assembly of the whistle in associated holes h formed in the upper part 2 (see FIGURES 12 to 16 and FIGURE 19).

[0019] As shown in FIGURES 1 and 2, the left end of the whistle 1 has a mouthpiece 4. This mouthpiece 4 leads to a sound chamber 5 of which part, designated 5a, is formed by a cavity in the lower part 3 and part, designated 5b, is formed by a cavity in the upper part 2. This sound chamber exhausts upwardly through a vent 6, as shown in FIGURES 1 and 2. The outlet of this vent is formed by a vertical surface 6a, a horizontal surface 6b and an inclined surface 6c which is of tapering form in section to provide a cutting edge or splitter 9. A central, transverse hole 7 is provided in a solid part of the whistle body at the right end as viewed in FIGURES 1 and 2 so that it does not communicate with the sound chamber 5, the hole being defined by opposed semi-cylindrical grooves formed in the upper and lower parts 2 and 3. This hole is used for a lanyard or cord (not shown) so that the whistle can be carried on the user's neck or wrist in well known manner.

[0020] In order to understand the internal structure of the whistle 1, the upper part 2 and the base 3 are shown in some detail in FIGURES 12 to 19 and 4 to 11 respectively.

[0021] As previously mentioned, the sound chamber 5 of the whistle 1 is defined by a cavity 5a in the lower part 3 of the whistle body and a cavity 5b in the upper part 2. These cavities are in register and together define a generally cylindrical chamber. The chamber communicates with the mouthpiece 4 through a passage defined in the body of the whistle by a longitudinal channel 10 formed in the upper part 2, the mouth of the channel being closed by a ramp 8 formed on the lower part 3. This ramp acts to direct the airflow produced by blowing the whistle into the sound chamber 5.

[0022] As shown in FIGURES 9 and 10 the part 5a of the sound chamber in the lower part 3 is bounded on three sides by a raised rib r. During manufacture this rib seats in a receiving groove g on three sides of the part 5b of the sound chamber in the upper part 2 shown in detail in FIGURES 17 and 18.

[0023] The whistle body is preferably made of polycarbonate material. The two parts 2 and 3 of the body are secured together by ultrasonic welding, the pegs p and rib r engaged respectively in the holes h and groove g acting as melt points to ensure an airtight connection.

[0024] It is to be noted that the compact design of the whistle shown in the drawings is the result of 9 to 12

months hard research by the Applicant in an attempt to optimise the various parameters involved in order to construct a compact whistle having a very specific narrow frequency bandwidth.

[0025] A very important feature of the whistle 1 is the form of the airflow-directing ramp 8 (see FIGURES 4, 5, 6 and 7). The ramp has a rectangular inclined surface which for ease of illustration is marked with crossing diagonal lines in FIGURES 5 and 7. The ramp 8 directs air from the mouthpiece 4 into the sound chamber 5 and the important new feature of the ramp is the angle of incline α marked as $4^\circ 30'$ in FIGURE 4. It is to be emphasised that this angle is a radical departure from the normal angle for the industry. A usual angle for α would be between 8° and 12° . The effect of reducing angle α is to reduce compression of the air blown into the mouthpiece at the point of exit to the cutting edge or splitter 9 (see FIGURE 12). It is the air striking the cutting edge or splitter 9 that is responsible for generating the unique sound of the whistle 1 and it has been found that reducing the compression of the air by reducing angle α will create a more even distribution of air and prevents some frequency fluctuation. Reduced compression has been achieved by choosing α to be within the range $3^\circ 30'$ to 5° , at least with a compact design of whistle 1 as shown in the FIGURES of the drawings. Experiments conducted led to an optimum angle for α of $4^\circ 30'$ in this particular case. It is believed that angle α would need to be adjusted for other designs within the limits of about $3^\circ 30'$ to 5° as stated.

[0026] Furthermore, the angle β of the cutting edge or splitter 9 is shown in FIGURE 12 as being 55° ; the industry standard being 38° for β . The effect of increasing the cutting edge or splitter angle β is to enable exhausting air from the sound chamber 5 to escape rapidly, thereby reducing any back pressure in the sound chamber. It is back pressure in the sound chamber which can create stalling and silencing of the whistle under pressure. A steep angle of greater than 38° (preferably 55° with the whistle parameters as shown in the drawings) more particularly acting in conjunction with angle α enables a more rapid escape of air from the whistle 1 through the vent 6 than has previously been achieved, reducing back pressure and preventing stalling/silencing the whistle under pressure. A marked improvement in performance over known whistles is provided by choosing an angle for β lying in the range about 45° to 55° even though a 55° angle would seem to achieve an optimum effect in conjunction with the other particular parameters of the whistle 1 as shown in the drawings.

[0027] Additionally, it is to be noted that the end of the ramp 8 adjacent the sound chamber 5 is formed with an inclined transverse face 8a (see FIGURES 4, 5 and 6), the angle of inclination being 45° and the length of the face being .05" (.127 cm). Although it has been found that the variation in this angle of the face 8a is less critical than the variation in angle α and angles β it still has a marked effect on the frequency fluctuation provided

by the whistle. The correct selection for this angle of face 8a allows a greater percentage of air as possible into the sound chamber 5, the effect being to create a constant frequency throughout the range.

[0028] The variation of angle α , β and the angle of the face 8a at the end of the ramp can all be varied in a selected way to provide an improved whistle.

[0029] Of additional importance is the ratio between the diameter D (see FIGURES 5 and 13) of the sound chamber 5 and the length L of the sound chamber. The diameter D and length L should be within 10% of one another. Alteration of the ratio of the diameter to the length of the sound chamber outside of these limits may have a dramatic effect on the frequency fluctuation when the whistle 1 is blown under pressure. Once this ratio is exceeded, the frequency of the whistle may move from a very tightly controlled 4200 hertz to 4600 hertz range (fluctuating range of 400 hertz only) up to a fluctuating range 800 hertz and sometimes more dependent upon the overall length of the sound chamber 5 selected.

[0030] The dog whistle 1 shown in the FIGURES of the drawings produces a very narrow frequency bandwidth of 400 hertz, under normal lung pressure, and has an average frequency of 4500 hertz with a maximum 4600 hertz and a minimum of 4200 hertz.

[0031] The physical features of the whistle 1 as afore-described have been specifically designed in order not only to create the narrow frequency band referred to but also to enable the whistle to produce frequencies falling within this range regardless of whether the whistle is blown softly or very hard. It is possible to blow the whistle 1 very hard without stalling or silencing the whistle as the blowing pressure increases.

[0032] It is believed that the general form of the remaining parts of the whistle 1 not described will be generally self-explanatory when the drawings are viewed in more detail.

[0033] It is believed that the particular dimensions of the whistle (in particular the overall length and width) are particularly advantageous resulting in a very compact, effective whistle.

Claims

1. A whistle (1) comprising a body (2,3) provided with a mouthpiece (4) having an air passage (10,8) which communicates with a sound chamber (5) provided in the body (2,3), the chamber (5) being provided with a vent (6) in which is located an air flow responsive means (9) for producing a whistling sound arranged so that air blown into the chamber 5 through the mouthpiece (4) is directed to said means (9) as it is vented from the chamber (5), characterised in that the sound chamber (5) is shaped and dimensioned to produce sound having a narrow frequency bandwidth of less than 800

hertz and in which the sound chamber (5) is generally cylindrical in shape and its diameter is within about 10% (ten percent) of the axial length of the chamber (5) and in which there is provided air flow directing means (8) comprising an operative surface inclined at an angle (α) in the range of about 3 degrees 30 minutes to about 5 degrees and in which an angle (β) of an air splitter (9) lies within a range of about 45 degrees to 55 degrees and the operative surface is provided at an end adjacent the sound chamber with a face (8a) having an angle of 45 degrees + or - 5 degrees being of such length to avoid stalling of the whistle whilst restricting variation of the frequency as the blowing pressure is varied to within said bandwidth.

2. A whistle (1) as claimed in Claim 1 in which the sound chamber (5) is shaped and dimensioned to produce sound having a narrow frequency bandwidth lying in the range of about 400 to 800 Hertz.
3. A whistle (1) as claimed in Claim 2 in which the sound chamber (5) is shaped and dimensioned to produce sound having a narrow frequency bandwidth of about 400 Hertz + or -20 hertz.
4. A whistle as claimed in any one of the preceding claims in which the said angle (α) is 4 degrees 30 minutes + or - 10 minutes.
5. A whistle as claimed in any one of the preceding claims in which said angle (β) of the air splitter is 55 degrees.

Patentansprüche

1. Pfeife (1) umfassend einen Körper (2,3), der mit einem Mundstück (4) mit einem Luftdurchgang (10,8) versehen ist, der mit einer in dem Körper (2,3) vorgesehenen Schallkammer (5) kommuniziert, wobei die Kammer (5) mit einem Auslass (6) versehen ist, in dem ein auf eine Luftströmung ansprechendes Mittel zur Erzeugung eines Pfeiftones so angeordnet ist, dass in die Kammer (5) durch das Mundstück (4) eingeblasene Luft zu dem Mittel (9) geleitet wird, wenn sie aus der Kammer (5) abgegeben wird, **dadurch gekennzeichnet, dass** die Schallkammer (5) so geformt und dimensioniert ist, dass ein Schall erzeugt wird, der eine schmale Frequenzbandbreite von weniger als 800 Hz hat, und dass die Schallkammer (5) eine im allgemeinen zylindrische Form hat und sein Durchmesser innerhalb von etwa 10% (zehn Prozent) der axialen Länge der Kammer (5) liegt, und dass ein Luftströmungs-Leitmittel (8) vorgesehen ist, das eine Wirkoberfläche aufweist, die unter einem Winkel (α) in dem Bereich von etwa 3 Grad 30 Minuten bis 5 Grad geneigt ist, und bei dem

ein Winkel (β) eines Luftteilers (9) in einem Bereich von etwa 45 Grad bis 55 Grad liegt, wobei die Wirkoberfläche an einem Ende neben der Schallkammer mit einer Stirnfläche (8a) versehen ist, die einen Winkel von 45 Grad + oder - 5 Grad und eine solche Länge hat, dass ein Abwürgen der Pfeife vermieden wird, während die Schwankung der Frequenz mit einer Schwankung des Anblasdrucks innerhalb der genannten Bandbreite begrenzt wird.

2. Pfeife (1) wie in Anspruch 1 beansprucht, bei der die Schallkammer (5) so geformt und dimensioniert ist, dass sie einen Schall mit einer engen Frequenzbandbreite erzeugt, die in dem Bereich von etwa 400 bis 800 Hz liegt.
3. Pfeife (1) wie in Anspruch 2 beansprucht, bei der die Schallkammer (5) so geformt und dimensioniert ist, dass sie einen Schall mit einer engen Frequenzbandbreite von etwa 400 Hz + oder -20 Hz hat.
4. Pfeife nach einem der vorhergehenden Ansprüche, bei der der Winkel (α) 4 Grad 30 Minuten + oder - 10 Minuten beträgt.
5. Pfeife nach einem der vorhergehenden Ansprüche, bei der der Winkel (β) des Luftteilers 55 Grad ist.

Revendications

1. Sifflet (1) comprenant un corps (2,3) pourvu d'une embouchure (4) ayant un passage d'air (10,8) qui communique avec une chambre de résonance (5) prévue dans le corps (2,3), la chambre (5) étant pourvue d'un orifice (6) dans lequel se trouve un dispositif responsable du souffle d'air (9) pour produire un sifflement disposé de manière à ce que l'air soufflé dans la chambre (5) à travers l'embouchure (4) soit dirigé vers ledit dispositif (9) lorsqu'il est déchargé à partir de la chambre (5), **caractérisé en ce que** la chambre de résonance (5) est formée et dimensionnée pour produire un son ayant une largeur de bande de fréquences étroite inférieure à 800 hertz et dans lequel la chambre de résonance (5) est généralement de forme cylindrique et son diamètre est inférieur à environ 10 % (dix pour cent) de la longueur axiale de la chambre (5) et dans lequel un dispositif de direction du souffle d'air (8) est prévu comprenant une surface opératoire inclinée à un angle (α) compris dans la plage allant d'environ 3 degrés 30 minutes à environ 5 degrés et dans lequel un angle (β) d'un répartiteur d'air (9) est compris dans une plage allant d'environ 45 degrés à 55 degrés et la surface opératoire est prévue sur une extrémité adjacente à la chambre de résonance avec une face (8a) ayant un angle de 45 degrés, + ou - 5 degrés, d'une longueur telle qu'elle permette

d'éviter le blocage du sifflet tout en restreignant la variation de la fréquence lorsque la pression de soufflage varie à l'intérieur de ladite bande de fréquences.

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2. Sifflet (1) selon la revendication 1, dans lequel la chambre de résonance (5) est formée et dimensionnée pour produire un son ayant une largeur de bande de fréquences étroite comprise dans la plage allant d'environ 400 à 800 hertz.

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3. Sifflet (1) selon la revendication 2, dans lequel la chambre de résonance (5) est formée et dimensionnée pour produire un son ayant une largeur de bande de fréquences étroite d'environ 400 hertz, + ou - 20 hertz.

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4. Sifflet selon l'une quelconque des revendications précédentes dans lequel ledit angle (α) est de 4 degrés 30 minutes, + ou - 10 minutes.

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5. Sifflet selon l'une quelconque des revendications précédentes dans lequel ledit angle (β) du répartiteur d'air est de 55 degrés.

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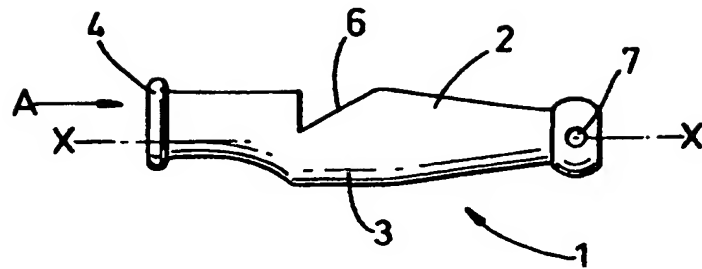


Fig. 1

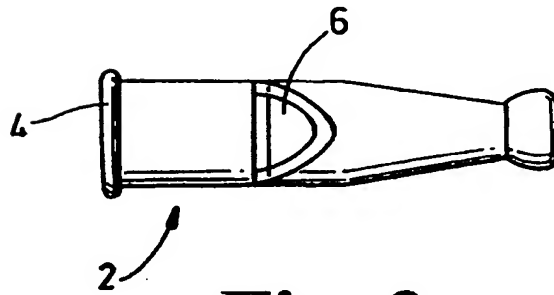


Fig. 2

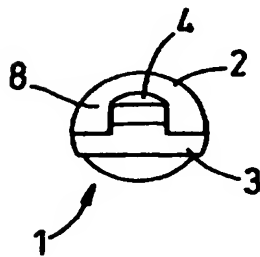


Fig. 3

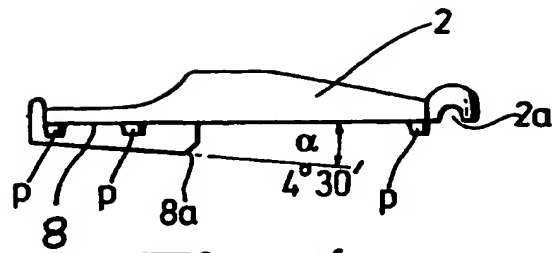


Fig. 4

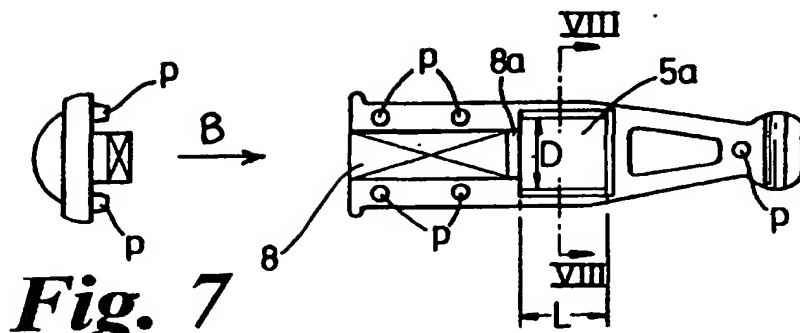


Fig. 5

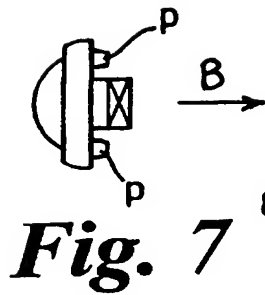


Fig. 7



Fig. 8

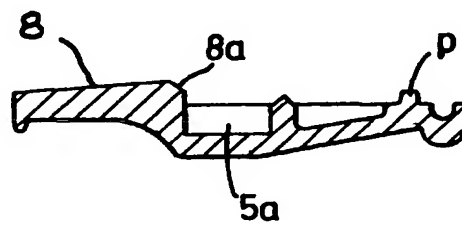


Fig. 6

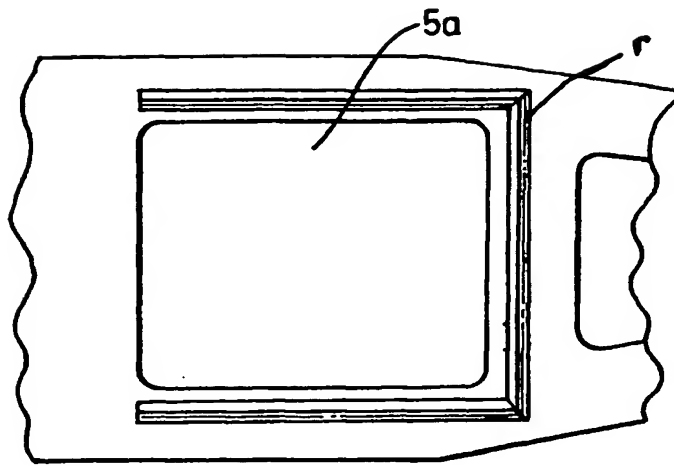


Fig. 9

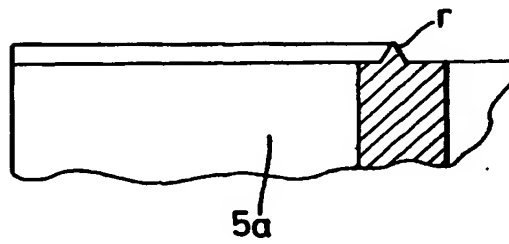


Fig. 10

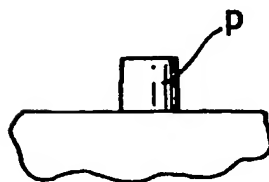


Fig. 11

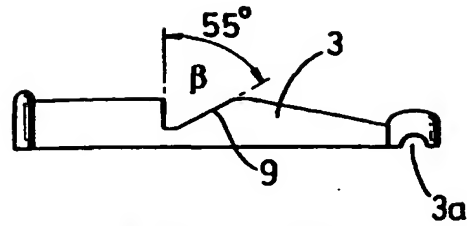


Fig. 12

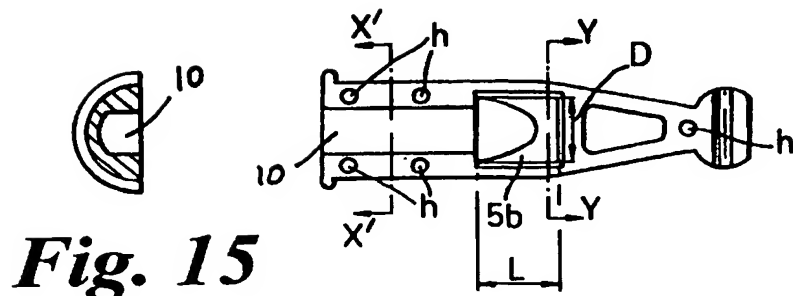


Fig. 13



Fig. 16

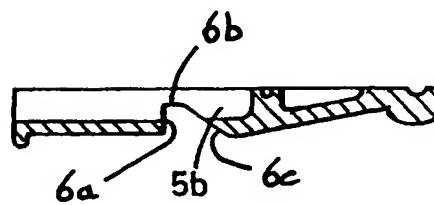


Fig. 14

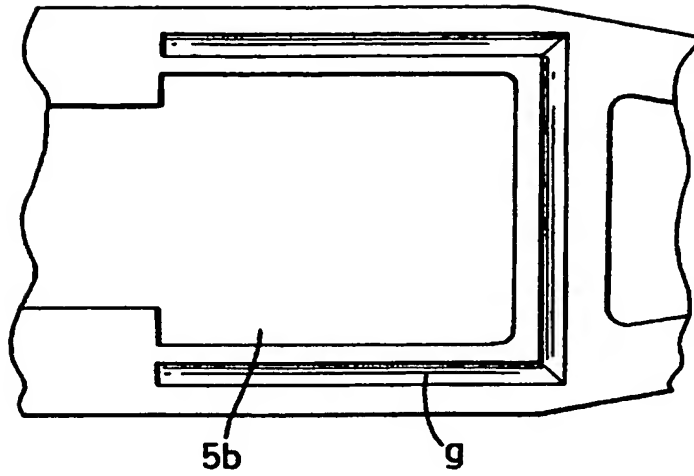


Fig. 17



Fig. 18

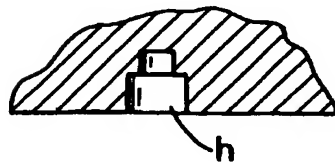


Fig. 19